AMENDMENTS TO THE CLAIMS:

Please amend the claims as follows:

Claim 1 (Currently Amended) A dosing device [[(1)]] for feeding an infusion product comprising:

a rotary drum type conveyor means [[(2)]] positioned between a web [[(3)]] of filter material and a hopper [[(4)]] for containing the infusion product; the rotary drum type conveyor means [[(2)]] having a plurality of radial dosing cells [[(5)]] made therein ~~in them~~ for containing the infusion product and in each cell is a sliding ~~which there slide piston type~~ dosing piston ~~means~~ [[(6)]]; each ~~dosing~~ piston [[(6)]] being driven axially by respective eccentric cam actuating means [[(7)]] between two end positions, one position corresponding to ~~of which corresponds to~~ a top dead centre [[(PMS)]] where each dosing cell [[(5)]] faces the hopper [[(4)]] in order to receive a quantity of the infusion product, and the other position corresponding to ~~corresponds to~~ a bottom dead centre [[(PMI)]] where ~~the~~ each dosing cell [[(5)]] faces the web [[(3)]] of filter material in order to discharge the quantity of infusion product onto the web [[(3)]] of filter material[[;]] ~~the device being characterized in that~~

wherein between the cam actuating means [[(7)]] and each piston [[(6)]] ~~there~~ are crank mechanisms [[(8)]] designed to act coaxially on the piston [[(6)]] in such a way as to enable the piston [[(6)]] to move in a direction that is perfectly aligned with a longitudinal axis [[(Z)]] of the respective dosing cell [[(5)]].

Claim 2 (Currently Amended) The device according to claim 1, ~~characterized in that~~ wherein the eccentric cam actuating means [[(7)]] comprise at

least one cam track [[(7a)]] in which there runs a cam follower [[(7b)]] for each piston [[(6)]]; the crank mechanisms [[(8)]] comprising a first crank [[(10)]] connected at one end to the cam follower [[(7b)]] and, at the other end, to a transmission shaft [[(11)]]; the transmission shaft [[(11)]] being rigidly attached to a first end of a second crank [[(13)]] that is in turn connected at its other end to a first end [[(14a)]] of a connecting control rod [[(14)]]; the connecting rod [[(14)]] being linked to the piston [[(6)]].

Claim 3 (Currently Amended) The device according to claim 2, ~~characterized in that a~~ wherein the first end of the second crank [[(13)]] is rigidly attached to the shaft [[(11)]] in such a way as to enable transmission of motion between the cam follower [[(7b)]] and the piston [[(6)]].

Claim 4 (Currently Amended) The device according to claim 2 or 3, ~~characterized in that~~ wherein the end of the second crank [[(13)]] that is linked to the connecting rod [[(14)]] is fork-shaped so as to hold the end of the connecting rod [[(14)]] on both sides; the connecting rod [[(14)]] being coupled with the second crank [[(13)]] through a first pin [[(16)]] that passes through a respective hole [[(17)]] made in said ~~forked~~ fork-shaped end.

Claim 5 (Currently Amended) The device according to claim 4, ~~characterized in that~~ wherein the connecting rod [[(14)]] is linked to the piston [[(6)]] by a second, transversal pin [[(18)]] which is housed in a respective hole [[(19)]] made in the piston [[(6)]] and which engages the respective end of the connecting rod [[(14)]].

Claim 6 (New) The device according to claim 1, wherein crank mechanisms include a first crank and a second crank, and

wherein the end of the second crank connected to the connecting rod acts coaxially on the piston.

Claim 7 (New) A dosing device for feeding an infusion product comprising:

a rotary drum type conveyor means positioned between a web of filter material and a hopper for containing the infusion product; the drum type conveyor means having a plurality of radial cells made therein for containing the infusion product and in each cell there is a sliding dosing piston; each piston being driven axially by respective eccentric cam actuating means between two end positions, one position corresponding to a top dead centre where each dosing cell faces the hopper in order to receive a quantity of the infusion product, and the other position corresponding to a bottom dead centre where the dosing cell faces the web of filter material in order to discharge the quantity of infusion product onto the web of filter material, the eccentric cam actuating means comprising at least one cam track in which there runs a cam follower for each piston; between the actuating means and each piston there being crank mechanisms designed to act coaxially on the piston in such a way as to enable the piston to move in a direction that is perfectly aligned with a longitudinal axis of the respective dosing cell, said crank mechanisms comprising a first crank and a second crank, the first crank being connected at one end to the cam follower and, at the other end, to a transmission shaft; the transmission shaft being rigidly attached to a first end of the second crank that is in

turn connected at its other end to a first end of a connecting control rod; the connecting rod being linked to the piston,

wherein the end of the second crank that is linked to the connecting rod is fork-shaped so as to hold the end of the connecting rod on both sides; the connecting rod being coupled with the second crank through a first pin that passes through a respective hole made in said fork-shaped end, and

wherein the end of the second crank that is connected to the connecting rod acts coaxially on the piston.

New Claim 7

New claim 7 recites similar features of claims 1-4 and new claim 6. Konig et al. ('342) does not disclose that the end of the second crank that is linked to the connecting rod is fork-shaped so as to hold the end of the connecting rod on both sides, and does not also disclose that the connecting rod is coupled with the second crank through a first pin that passes through a respective hole made in said fork-shaped end. In fact, as already cited, Konig et al. ('342) shows that the crank 113 is eccentrically connected to the respective connecting rod 114 which drives the piston 43, and such a connection between the crank 113 and connecting rod 114 does not comprise any fork-shaped ends. Moreover, such an eccentric linkage would product a bending or flexion action on the connecting rod 114 with respect to a direction along which the latter translates. Such a bending action, when transmitted to the piston, would cause problems for the sliding motion of the piston, so leading to a teaching-away from the underlying technical problem of the present application.

Further, with respect to the combination of Konig et al. ('048) and Romagnoli ('808), as discussed above, the combination fails to disclose a crank mechanism including a first and a second crank for moving the pistons in a radial direction. The combination also fails to disclose such fork-shaped end.

Since none of Konig et al. ('342), Romagnoli ('808) and Konig et al. ('048) disclose that the end of the second crank that is linked to the connecting rod is fork-shaped so as to hold the end of the connecting rod on both sides, new claim 7 is considered to be non-obvious over the prior art.

In view of the above, none of Konig et al. ('342) does not anticipate new claim 7 of the application. Further, Konig et al. ('048) and Romagnoli ('808), in combination, do not render new claim 7 obvious.